



OCCASION

This publication has been made available to the public on the occasion of the 50th anniversary of the United Nations Industrial Development Organisation.



DISCLAIMER

This document has been produced without formal United Nations editing. The designations employed and the presentation of the material in this document do not imply the expression of any opinion whatsoever on the part of the Secretariat of the United Nations Industrial Development Organization (UNIDO) concerning the legal status of any country, territory, city or area or of its authorities, or concerning the delimitation of its frontiers or boundaries, or its economic system or degree of development. Designations such as "developed", "industrialized" and "developing" are intended for statistical convenience and do not necessarily express a judgment about the stage reached by a particular country or area in the development process. Mention of firm names or commercial products does not constitute an endorsement by UNIDO.

FAIR USE POLICY

Any part of this publication may be quoted and referenced for educational and research purposes without additional permission from UNIDO. However, those who make use of quoting and referencing this publication are requested to follow the Fair Use Policy of giving due credit to UNIDO.

CONTACT

Please contact <u>publications@unido.org</u> for further information concerning UNIDO publications.

For more information about UNIDO, please visit us at www.unido.org

Final report

Hangzhou XiLing Refrigeration Electric Appliance CO.

Test Report

| Elaborated (编制) | by | Mr. Cai Dunbing | 黄家新 | April 8,98 |
|--------------------|----|-----------------|-----------------|------------|
| Checked (校对) | by | Mr. Wang Jun | 五路 | 10/4/1998 |
| Approved (批准) | by | Mr. Cai Haitao | المالية المالية | 15/4/98 |

Product Test Report

Refrigerator Model: BCD-188B

Test Standard: SYDG9802

Test Date: March 17, 98

Refrigerant: R600a

Product Test Lab of Development Center

1. Preface

The tests have been performed based on the following items in order to adjust the refrigerant circuit of XiLing BCD-188B with R600a as refrigerant:

- 1. Using special compressor
- 2. Adjusting the flow of capillary tube
- 3. Adjusting the filled quantity of refrigerant

The condenser and the evaporator have not been adjusted in the test and there are not many different types of compressors to choice, so the compare test of different compressors has not been performed. On account of the test conditions and the limited time, the experiment to fix the optimal flow of capillary tube has not been carried out.

We shall get following aims after the conversion experiment:

- 1. By optimizing the R600a refrigerant circuit, the energy consumption should near to or lower than that of R12 refrigerant circuit
- 2. To master the fundamental law of refrigerant circuit of BCD-188B with R600a as refrigerant

2. Test conditions and Test procedure

- 1. Test conditions: the lab and test have been carried out the principle of China standard GB8059.2-1995
- 2. Test procedure: we test the items as the following order:
- continuous running test at ambient temperature of t_a=32 °C
- Energy consumption at t_a =25 °C, $t_i \le 5$ °C
- Running test at low temperature t_a=16 °C
- Running test at high temperature t_a=32 °C

3. Technical data indicated

| compressor brand | National | type of compressor | QD100Y |
|-----------------------|------------|--------------------|--------|
| thermostat brand | FuShan | type of thermostat | WDF-29 |
| standard of capillary | 1.9X0.71mm | flow of capillary | 270L/h |
| evaporator No.1 | - | evaporator No.2 | - |
| condenser No.1 | .= | condenser No.2 | - |
| *the filled quantity | 36g | | |

Note: "-" stands the same to R12 refrigerant circuit

"*" stands the initial the filled quantity

4. Continuous running result

| ti cooler | tj freezer | turn of the filled quantity |
|-----------|------------|-----------------------------|
| -7.38 ℃ | -32.61 ℃ | 39g |

5. Energy consumption result

| | parameter | | | | | thermostat pos. | | | | | | | 5 | | |
|-------|-----------------------|-----|------|-----|-------|------------------------|-------|-------|---------|----------|-----|------------|-------|--------------|--|
| | cooler temperature(℃) | | | | | freezer temperature(℃) | | | | | | | | running time | |
| • | T10 | T11 | T12 | Tm | T1 | T2 | T3 | T4 | T5 | Т6 | T7 | Т8 | Th | (min) | |
| start | 6.8 | 5.1 | 4.0 | | -17.6 | -18.0 | -20.8 | -21.2 | -19.0 | -19.5 | | | -18.0 | 37 | |
| stop | 2.9 | 0.1 | -1.1 | 3.0 | -19.3 | -18.1 | -19.8 | -20.4 | -19.6 | -20.0 | | | | 53 | |
| w | working period | | | | 90min | | | wo | rking o | coeffici | ent | 41 percent | | | |
| avera | average running power | | | | 91 | W | | ene | ergy co | nsump | ion | 0.898KWh/d | | | |

6. Running test at low ambient temperature(ta=16 $^{\circ}$ C)

| para | parameter | | | | | | | thermostat pos. | | | | | | 4 | |
|--------|-----------------------|-----|-----|-----|--------|-------------------------|-------|---------------------|-------|-------|----|------------|-------|-------|--|
| | cooler temperature(℃) | | | | | freezer temperature(°C) | | | | | | | | | |
| | T10 | T11 | T12 | Tm | T1 | T2 | T3 | T4 | T5 | T6 | T7 | T8 | Th | (min) | |
| start | 6.4 | 5.5 | 4.7 | | -18.5 | -18.7 | -20,4 | -20.8 | -18.5 | -19.2 | | | -18.5 | 29 | |
| stop | 3.2 | 1.5 | 0.3 | 3.6 | -18.8 | -19.1 | -20.7 | -21.2 | -19.1 | -19.7 | | | | 85 | |
| wo | working period | | | | 114min | | | working coefficient | | | | 25percent | | | |
| averag | average running power | | | | | W | | energy consumption | | | | 0.569KWh/d | | | |

7. Running test at high ambient temperature(ta=32 °C)

| para | parameter | | | | | | | thermostat pos. | | | | | 5 | | |
|-----------------------|-----------|--------|--------|-------|------------------------|-------|---------------------|--------------------|-------|-------|------------|------------|-------|--------------|--|
| | cool | er tem | peratu | re(℃) | freezer temperature(℃) | | | | | | | | | running time | |
| | T10 | T11 | T12 | Tm | T1 | T2 | Т3 | T4 | T5 | T6 | <u>T</u> 7 | Т8 | Th | (min) | |
| start | 8.3 | - 6.1 | 4.9 | | -18.0 | -18.5 | -21.5 | -21.9 | -19.5 | -19.9 | | | | 41 | |
| stop | 4.9 | 1.6 | 0.3 | 4.4 | -18.2 | -18.7 | -21.6 | -21.9 | -19.8 | -20.2 | | | -18.0 | 39 | |
| working period | | | | 80n | nin_ | | working coefficient | | | | 51 percent | | | | |
| average running power | | | | | 91 | W | | energy consumption | | | | 1.119KWh/d | | | |

8. Analysis and discuss

- 1. According to the result of continuous running test, the filled quantity of refrigerant in the conversed refrigerant circuit is 48g.It is 40 percent of what the R12 refrigerant circuit needs.
- 2. The energy consumption of conversed refrigerant circuit is 10 percent lower than that of R12 refrigerant circuit. The working coefficient of two circuits is same, but the running power of R600a refrigerant circuit is smaller than that of R12's.
- 3. The working period of conversed refrigerant circuit becomes longer and the temperature differential between cooler and freezer is small and the energy consumption does not meet what we have expected, so all of above indicated that the refrigerant circuit can be optimized.